



# Space Fan2 : A Mechanical De-Orbiting Device System for Satellites

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# Our Team

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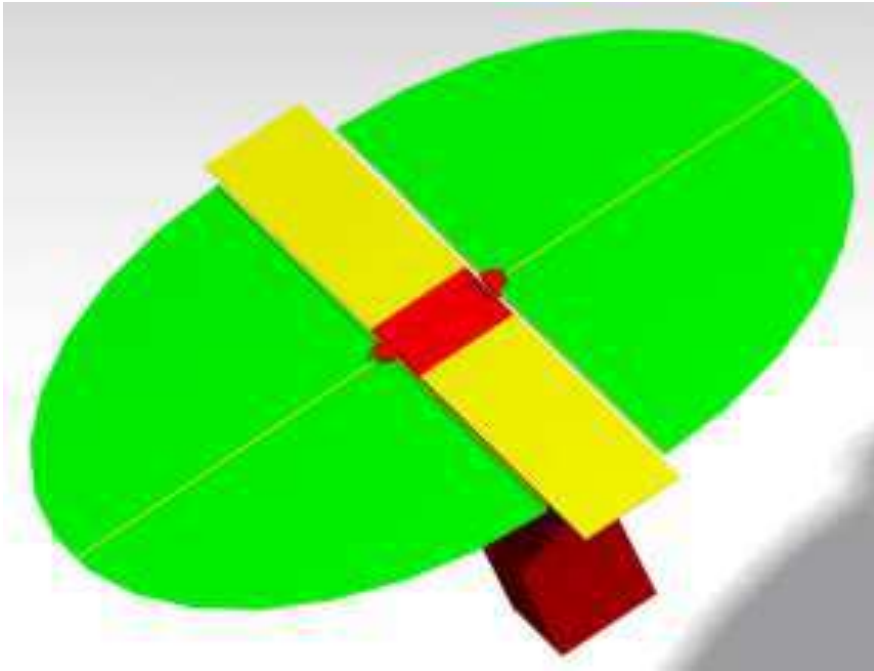
# Competition Requirements

- The device must be designed for the removal of a potentially non-cooperative lean satellite of 50 kg mass and maximum dimension of 1 meter. Total mass of a satellite and device can exceed 50 kg.
- The device will enable the satellite to re-entry within 11 years (i.e. one solar cycle) after activating. You can use any systems such as thruster, tether, membrane or electric propulsion.
- The device will be activated at 00:00:00 UTC, January 1, 2020 with the following orbit element:

<b>Semi Major Axis</b>	<b>7128 km</b>
<b>Inclination</b>	98.4 degree
<b>R.A.A.N</b>	30 degree
<b>Argument pf Perigee</b>	210 degree
<b>Mean Anomaly</b>	190 degree
<b>Atmosphere model</b>	Jacchia-Roberts

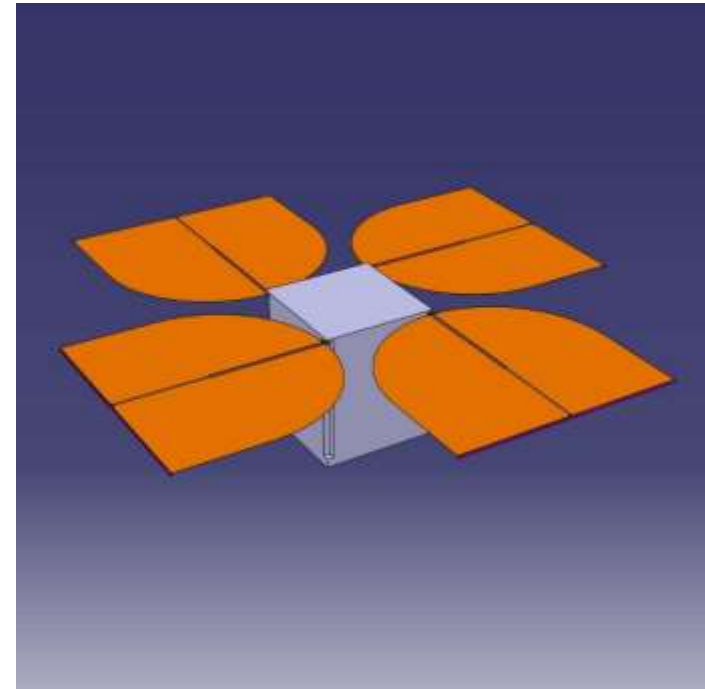
# Mission Overview

Space Fan 2 is based on last year's design Space Fan.



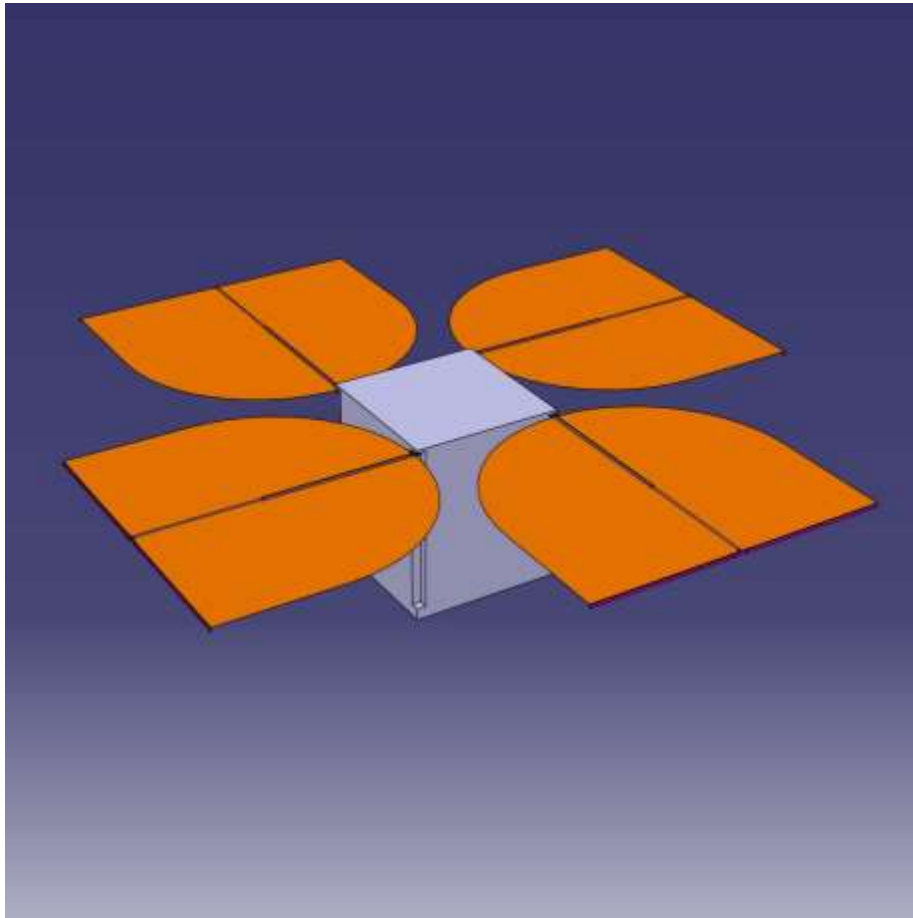
Space Fan

Space Fan 1 Available in UNISEC-GLOBAL website



Space Fan 2

# Mission Overview and Physical Layout



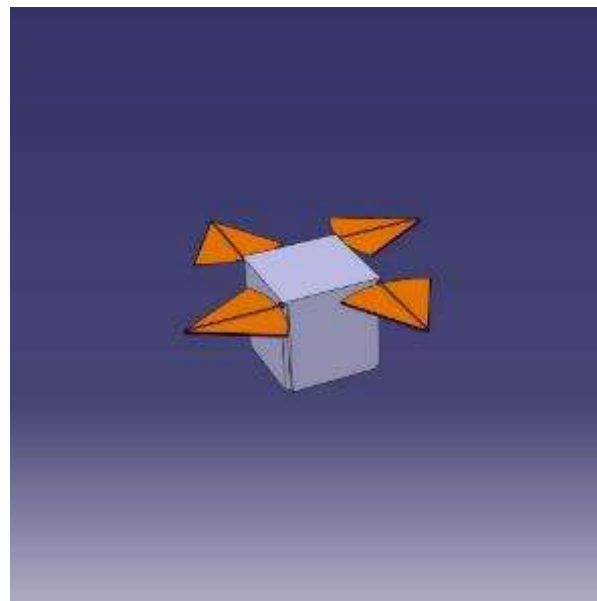
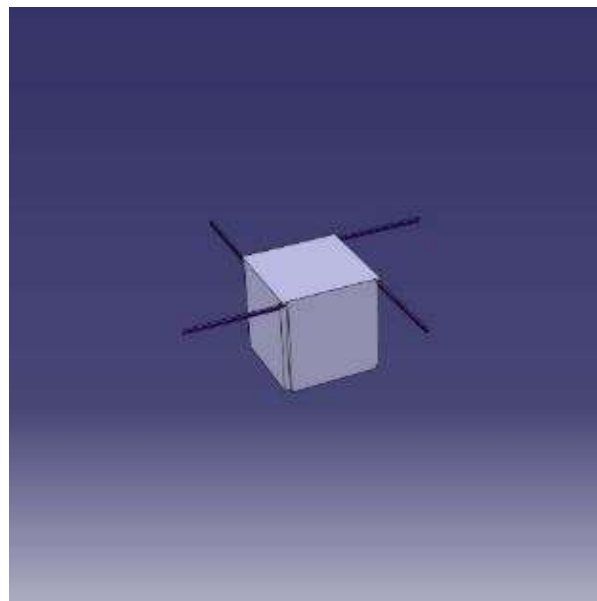
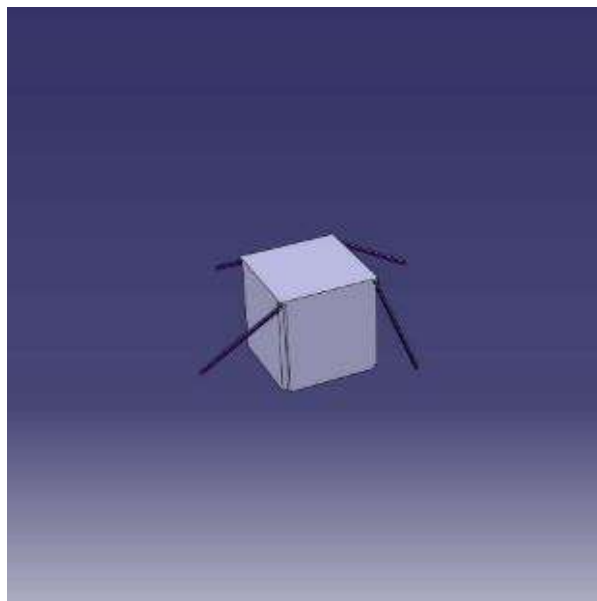
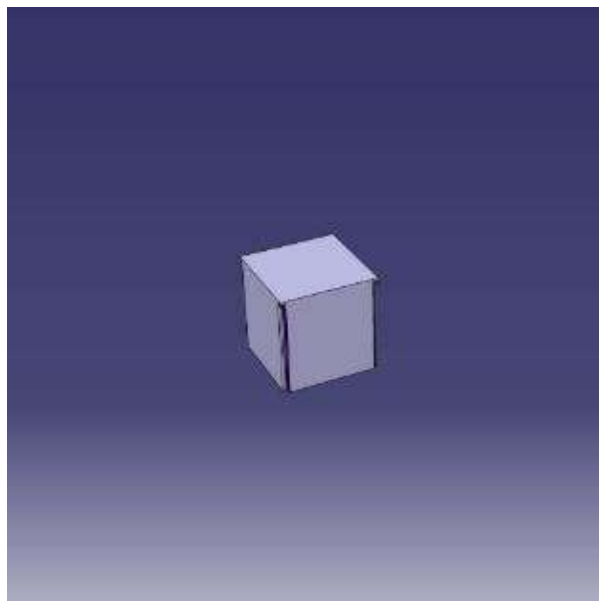
- Increasing drag area to decrease de - orbiting time
- Consist of 4 fans

Dimensions of satellite : 100 cm x 100 cm x 100 cm

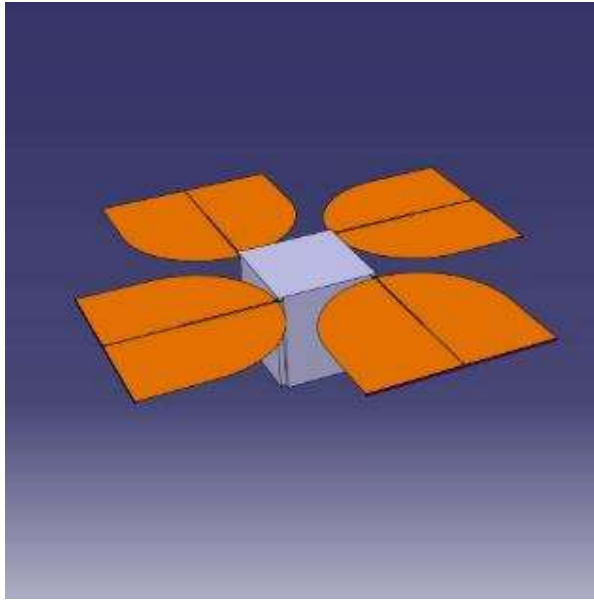
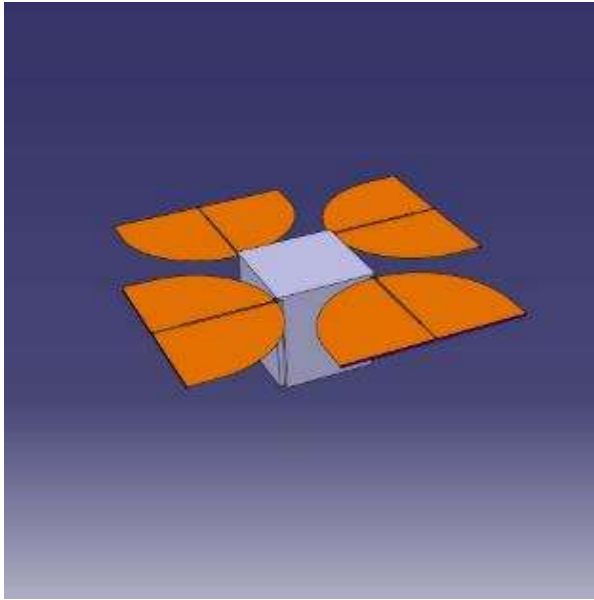
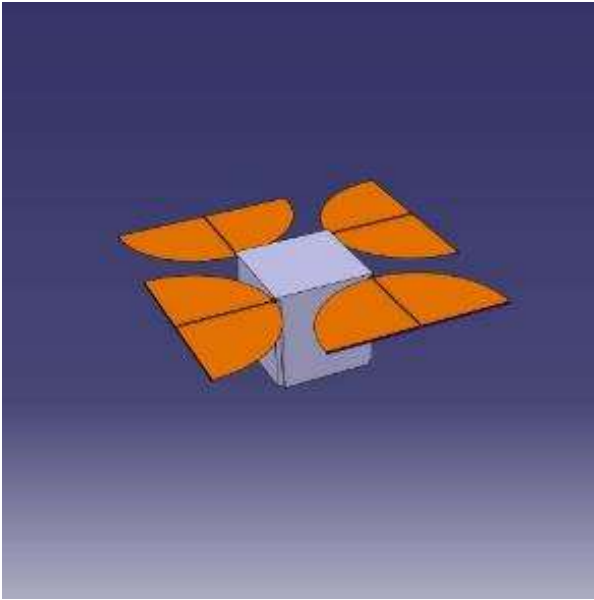
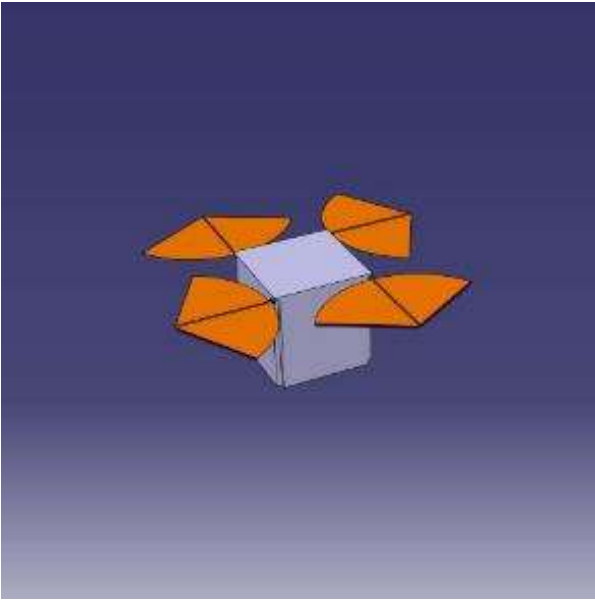
Total surface area of the sail is 10.804 m<sup>2</sup>.

Total Mass : 50 kg (include de-orbit system)

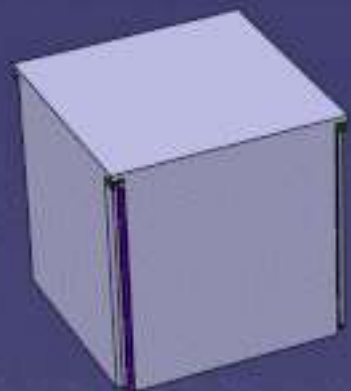
# Mechanical Deployment Design



# Mechanical Deployment Design

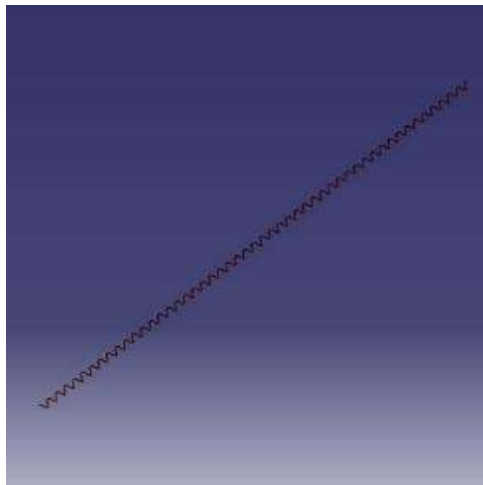




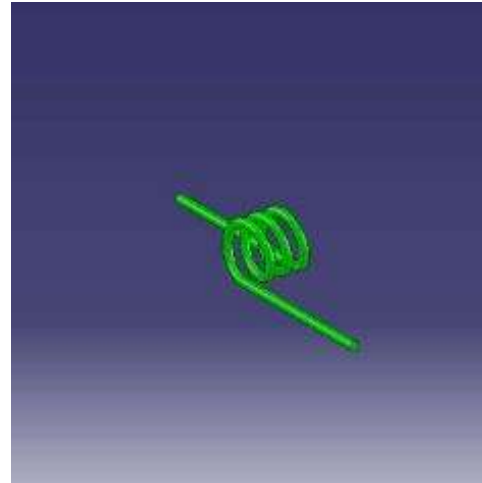


# Components

- **Compression Springs**

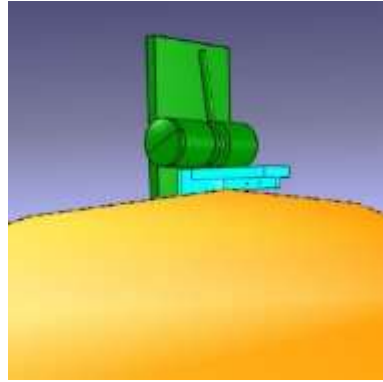
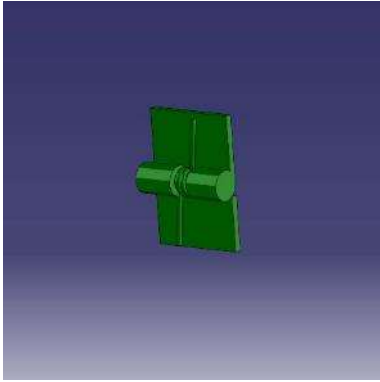


- **Torsion Spring**

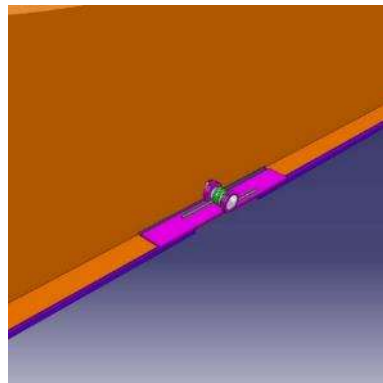
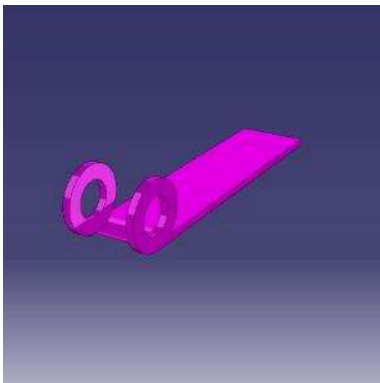


# Components

- **Body Hinge**

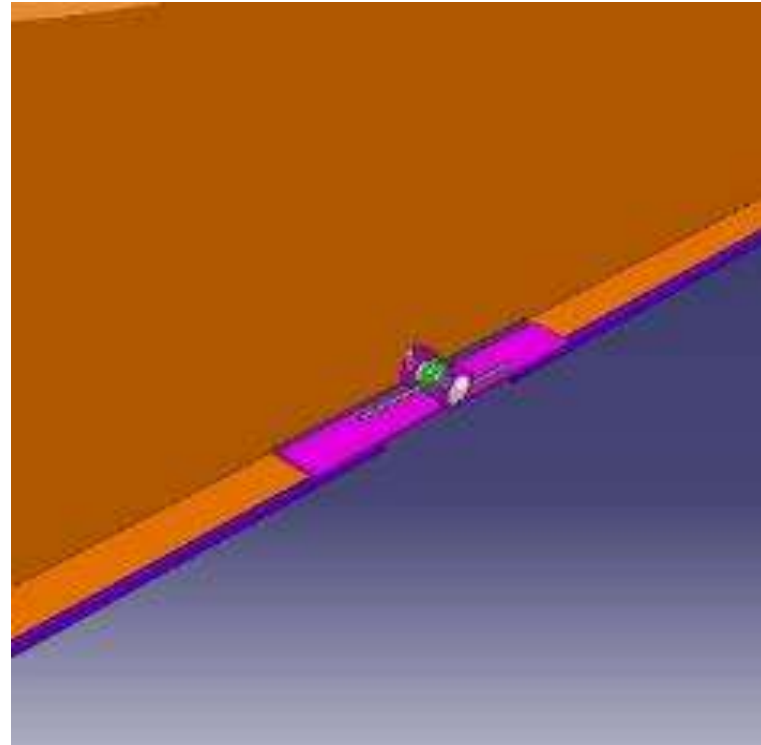
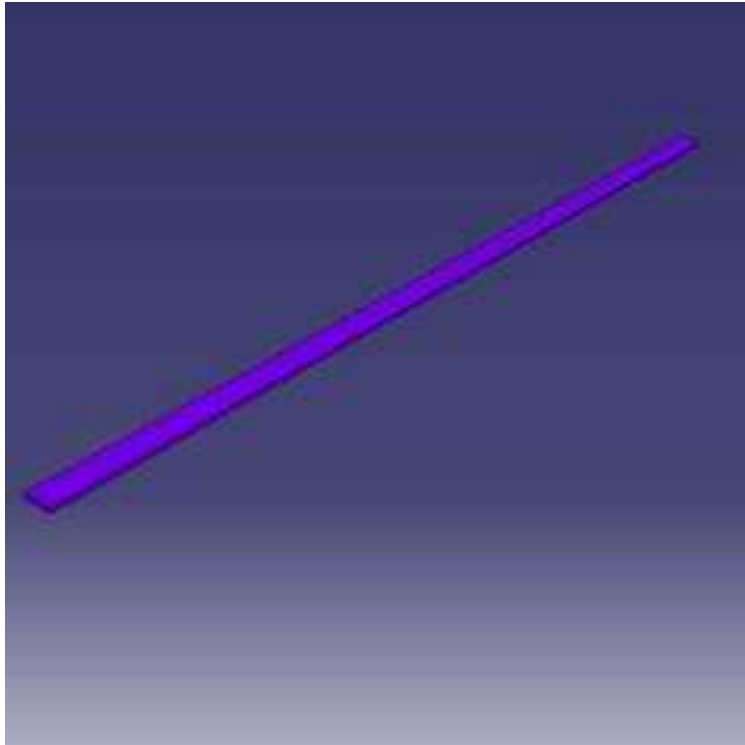


- **Sail Hinge**



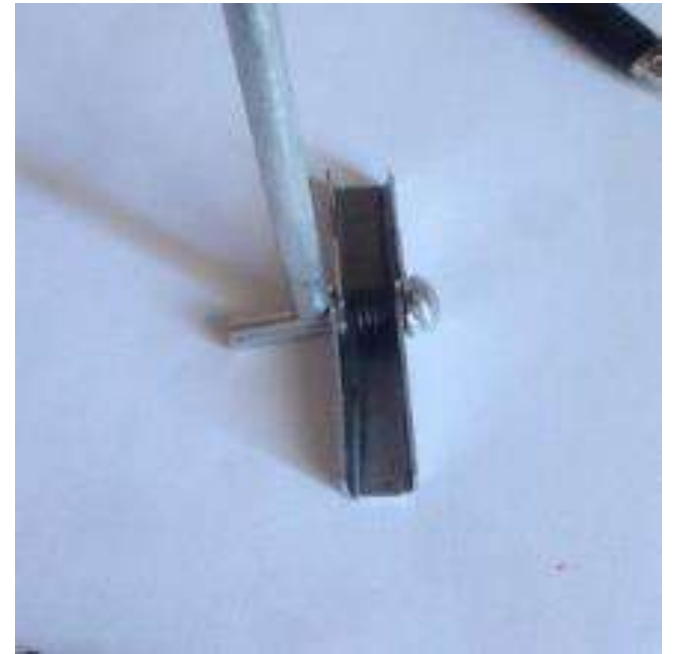
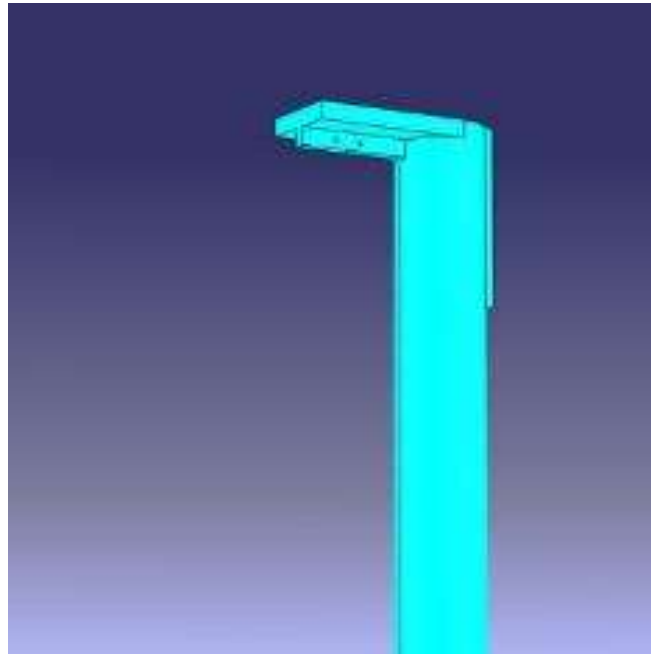
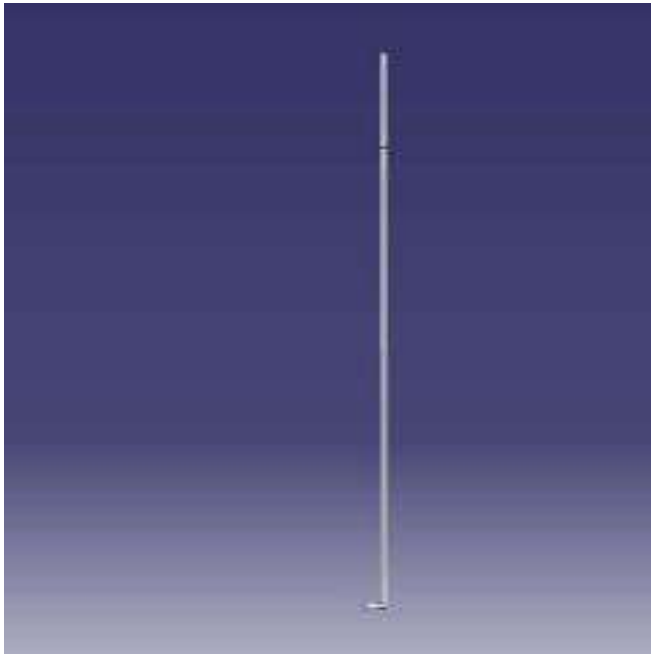
# Components

- Sail Wing



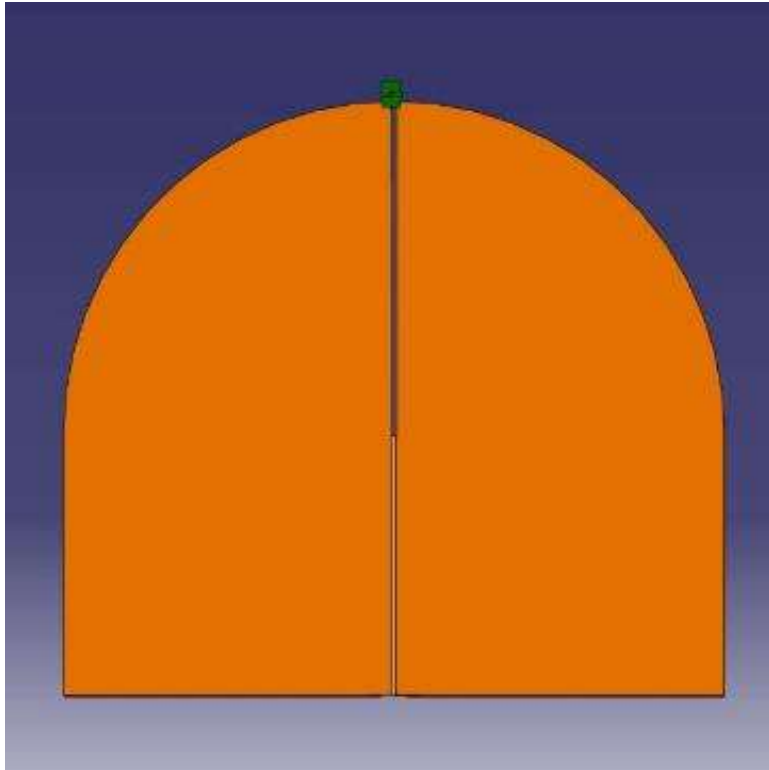
# Components

## Arm and Arm Housing



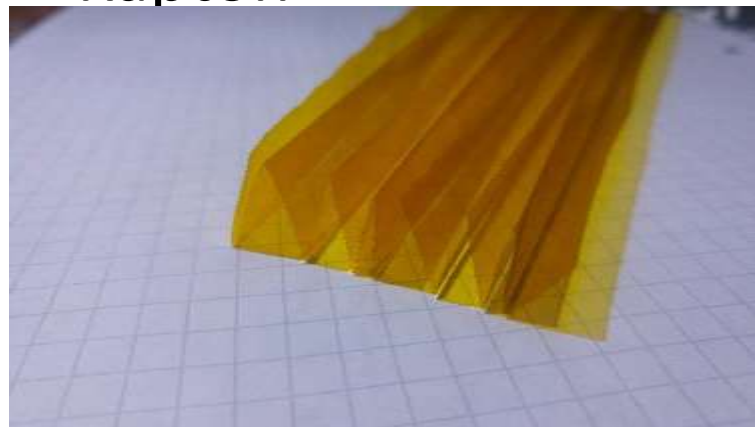
# Components

- Fan



# Sail Material Selection

- Kapton



Property	Unit	75μm	125μm
Ultimate Tensile Strength at 23°C, (73°F) at 200°C (392°F)	psi (MPa)	33,500(231) 20,000(139)	33,500(231) 20,000(139)
Density	g/cc	1.42	1.42
Tear Strength, Initial (Graves), N (lbf)		26.3 (1.6)	46.9 (1.6)

Property	Typical Value	Test Condition
Thermal Coefficient of Linear Expansion	20 ppm/°C (11 ppm/°F)	-14 to 38°C (7 to 100°F)
Specific Heat, J/g•K (cal/g•°C)	1.09 (0.261)	
Shrinkage, %		
30 min at 150°C	0.17	
120 min at 400°C	1.25	

- Mylar Film



Property	Unit	142μm
Tensile Strength MD	kpsi	28
Tensile Strength TD	kpsi	34
Elongation at Break MD	%	125
Elongation at Break TD	%	100

Property	Unit	142μm
Shrinkage MD (150°C)	%	1.5
Shrinkage TD (150°C)	%	1.0

# Sail Material Selection

- The values given in the tables are the results of the tests made by the manufacturer and the film features presented to the user. If we consider the characteristics of both films: Using 125 $\mu$ m Kapton HN Film will be more advantageous because although it is thin,
- Tensile strength more than Mylar,
- The amount of elongation is less than Mylar,
- It shrinks less with temperature regarding to Mylar,
- When we take into consideration that it is thin, the time we fold will fit easily between the wings in our design.



# Prototype Development Phase



# Prototype Development Phase



# Prototype Development Phase



# Mass Budget

Components	Mass(g)	%20 margin (g)	Predicted Mass (g)
Longbow(steel)	17.0316	3.40632	20.43792
Tube(aluminum)	170.1108	34.02216	204.133
Inner arm(aluminum)	112.062	22.4124	134.4744
Wing x 2 (aluminum)	109.8276	21.96552	131.7931
180 degree opening hinge part (aluminum)	5.358	1.0716	6.4296
180 degree opening bow (steel)	2.508	0.5016	3.0096
First hinge (steel)	121.1136	24.22272	145.3363
Kapton	5.0616	1.01232	6.07392
<b>TOTAL 4 Parts</b>	<b>2172.2928</b>	<b>434.4584</b>	<b>2606.75</b>

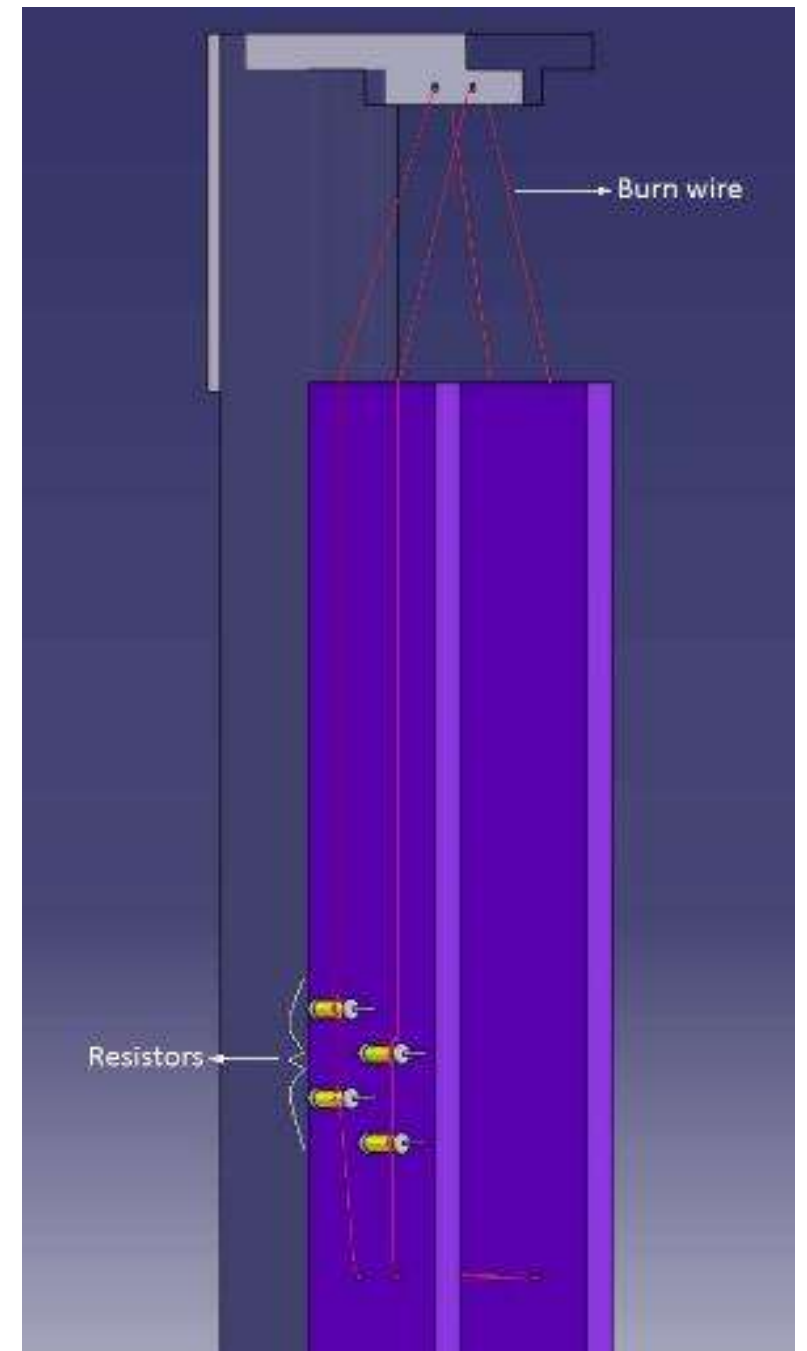
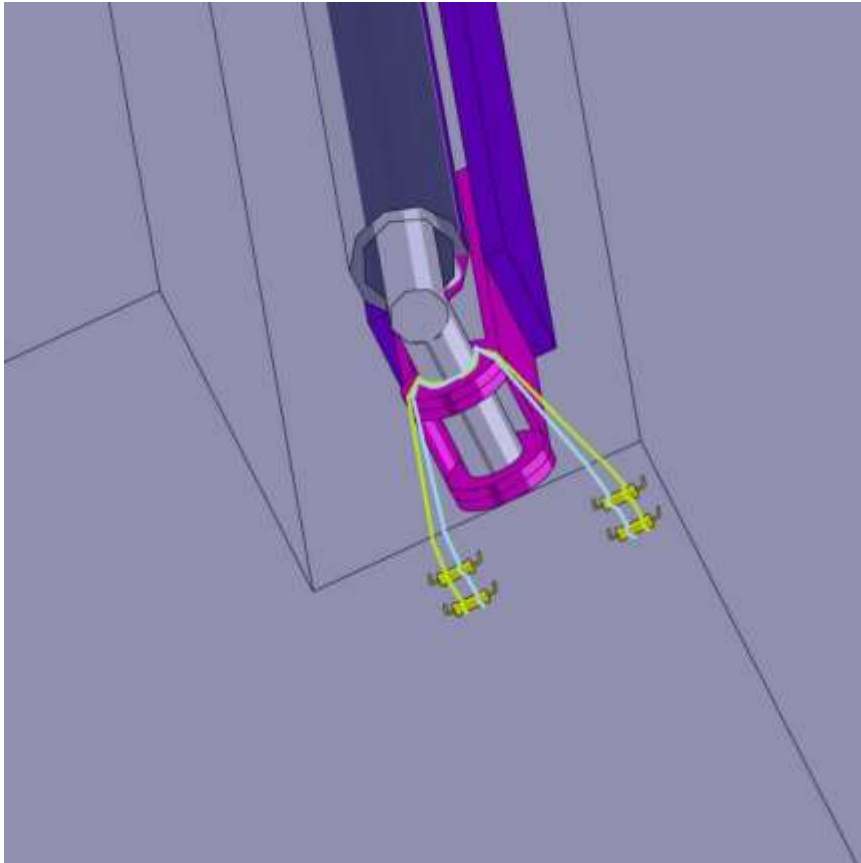
# Cost Budget

Components	Unit	Cost	Total
Springs	4 pieces	\$3	\$12
Spring hinge	8 pieces	\$2.5	\$20
Frame	4 pieces	\$5	\$20
Kapton/ Aluminum	11 m2	\$16.06	\$225
Screws	16 pieces	\$0.3	\$4.8
Fish line	8m	\$1	\$1
Total			<u>\$282.8</u>



# Activation System

- The opening of the sails will be provided by resistors. 2 fishline and 2 resistors for each fishline is used in every mechanism to ensure safety.



# Activation System

- The burn wire used for opening sails can withstand a weight of 37 kg. Since it is not possible to carry out the test for the duration of the task, the test was carried out for 7 days. No stretching or distortion was observed in the tests.



In the room conditions, it was observed that the resistance of the burn wire was burned with resistance at 5V, 10 ohms, 0.48 amperes and 7 seconds. Nichrome wire can be used instead of resistor.

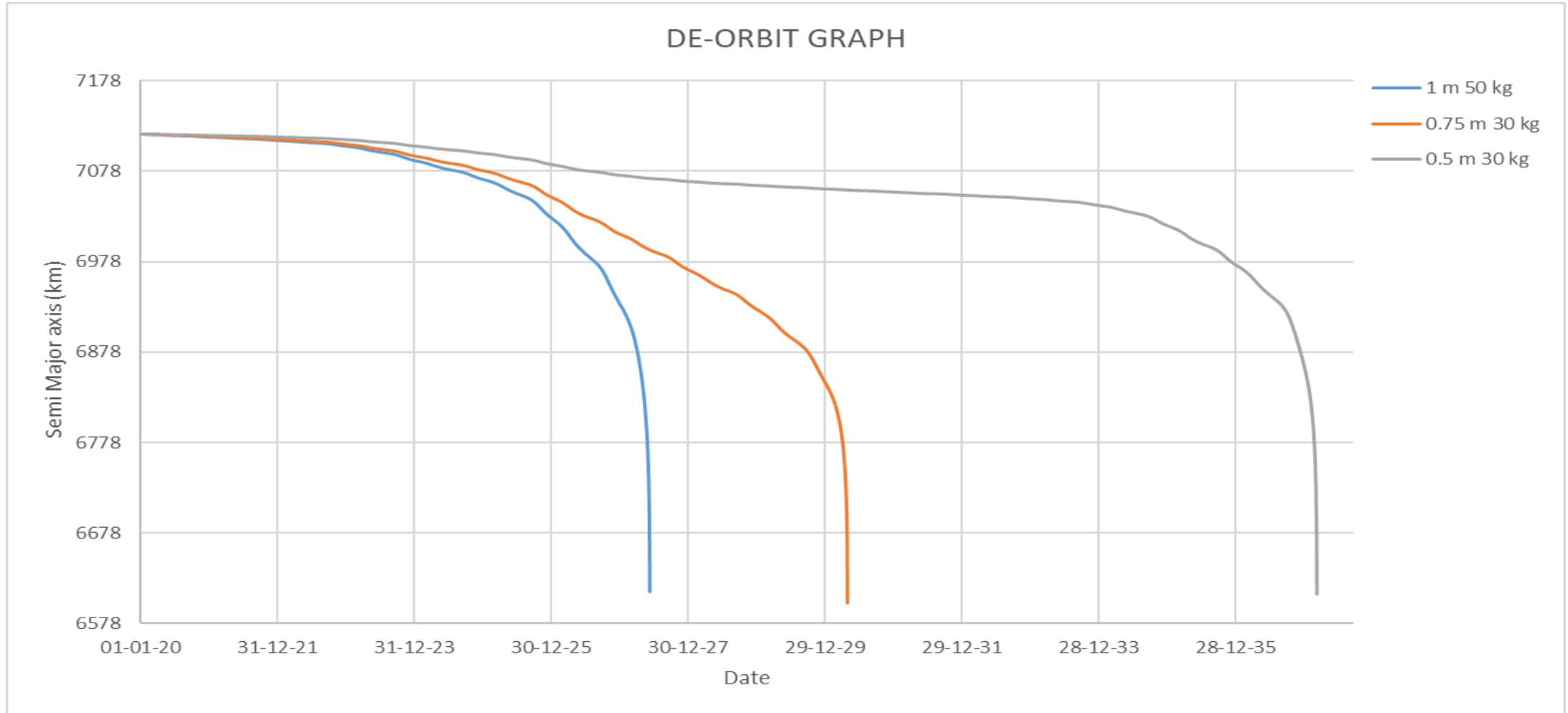
# Lifetime Analysis

Case	Height of the Satellite (meter)	Mass of the satellite (kg)	Drag Area of the Solution (m <sup>2</sup> )	Total Drag Area with Satellite (m <sup>2</sup> )	Lifetime (Days)	Lifetime (Years)
1	1	50	10.804	11.804	2716	7.4
2	1	52	10.804	11.804	2875	7.9
3	1	40	10.804	11.804	2281	6.2
4	0.75	30	6.077	7.077	3772	10.3
5	0.75	50	6.077	7.077	6313	17.3
6	0.5	50	2.701	3.701	16496	45.2
7	0.5	40	2.701	3.701	7524	20.6
8	0.5	30	2.701	3.701	6276	17.2
9	1	50	0	1		340

Lifetime analysis of a cubic satellite at 750 km altitude. Analysis made with System Tool Kit (STK)



# Lifetime Analysis



# Conclusion and Future work

- Simple and cheap
  - More effective at lower altitudes.
- Future work
- We will integrate a lock system for deployed parts to prevent oscillation of the deployed part with spring force.
  - We will design a better prototype

THANK YOU FOR YOUR ATTENTION

